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# A Nonlinear Galloping-Driven Triboelectric-Electromagnetic Hybrid Generator for Low-Speed Wind Energy Harvesting

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## Resources availability

### *Materials availability*

This study did not generate new, unique reagents.

### *Data and code availability*

The data and code that support the conclusions of this study are also available from the lead contact upon reasonable request.

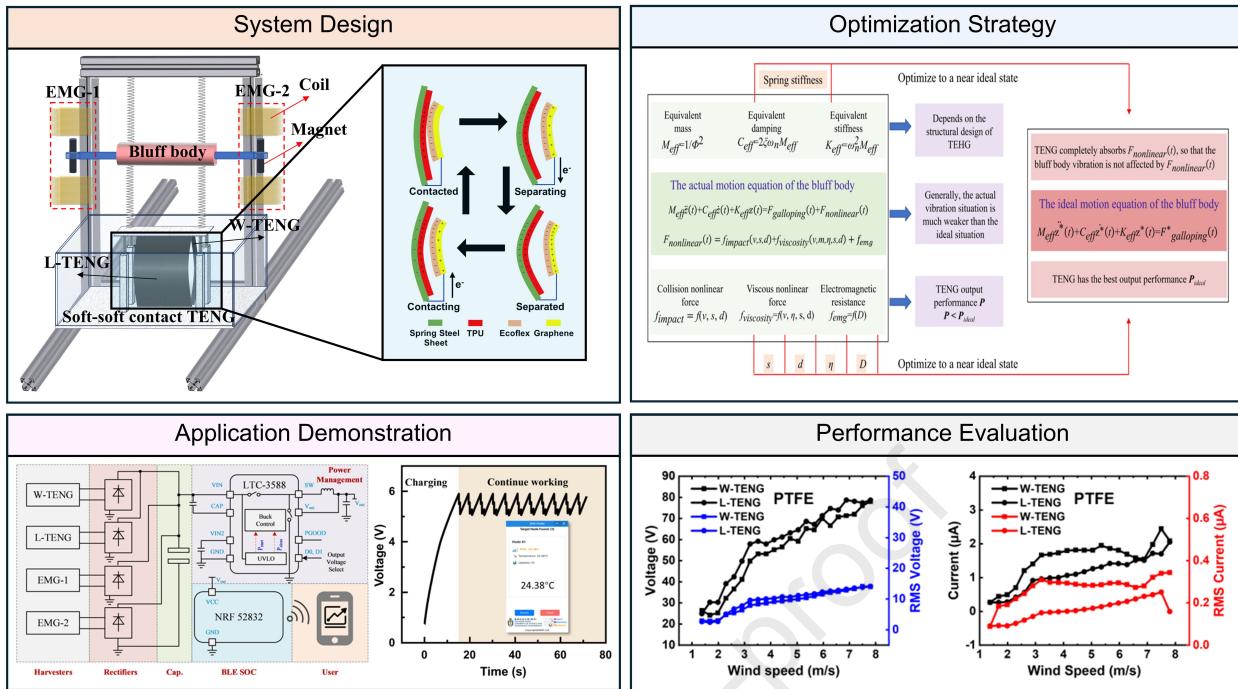
## Supporting Information

### *Author contributions*

X.K., P.L., and J.W. conceived the idea. X.K., G.H. and J.W. secured the funding. X.K. and P.L. designed the experiment, conducted the test and measurements, collected the data, and drafted the original manuscript. X.K., P.L., and Y.W. prepared the figures. Y.W. and Y.L. designed and developed the circuit boards. R.Z., Y.J., G.H., and Q.S. polished the manuscript. G.H. and J.W. supervised and guided the project.

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# 1 A Nonlinear Galloping-Driven Triboelectric-Electromagnetic Hybrid

## 2 Generator for Low-Speed Wind Energy Harvesting

3 \*\*\*

## 4 Abstract:

5 Triboelectric nanogenerators (TENGs) have emerged as a prominent technology  
6 for self-powered systems due to their remarkable outputs, low cost, and customizable  
7 structural designs. When integrated with flow-induced vibration (FIV), TENGs present  
8 a promising potential for harvesting low-speed wind energy. This study introduces a  
9 novel nonlinear galloping-driven triboelectric-electromagnetic hybrid generator (NG-  
10 TEHG) that can adapt to a wide range of wind speeds. We provide a comprehensive  
11 quantitative analysis of the nonlinear coupling effects between the TENG and bluff  
12 body galloping dynamics, revealing their impacts on the energy harvesting performance  
13 of the NG-TEHG. Our experimental results demonstrate that the NG-TEHG operates  
14 effectively within a wind speed range of 2.29–7.80 m/s, delivering a peak power of 10  
15 mW and an RMS power of 3.34 mW at a wind speed of 4.74 m/s. This achieved power  
16 is sufficient to continuously drive a wireless temperature sensing node, showcasing the  
17 NG-TEHG’s potential for practical applications. The hybrid system introduced in this  
18 study not only extends the range of wind energy harvesting but also provides a novel  
19 approach to galloping-based energy harvesting technologies.

20 **Keywords:** triboelectric nanogenerator (TENG); flow-induced vibration; galloping;  
21 hybrid energy harvesting; nonlinear coupling dynamics

## 1. Introduction

2 Low-speed wind energy is an abundant yet underutilized form of clean energy that  
 3 holds promise for sustainable power generation. Triboelectric nanogenerators (TENGs),  
 4 known for their excellent performance at low frequencies, strong scalability, and low  
 5 cost, offer an ideal solution for harvesting such energy to build self-powered wireless  
 6 sensing systems.<sup>1-3</sup> Notably, low-speed wind energy is typically treated as a steady fluid,  
 7 and when such a fluid flows over a structure, the fluid-structure interaction (FSI) may  
 8 induce self-excited vibration. This phenomenon, referred to as flow-induced vibration  
 9 (FIV), can be leveraged as an efficient mechanism for mechanical-to-electrical energy  
 10 conversion.<sup>4</sup> TENGs that operate based on the FIV principle, referred to as FIV-TENGs,  
 11 often exhibit stable and superior performance and have thus been widely employed for  
 12 harvesting flow-kinetic energy from wind, water, and other fluid environments.<sup>5</sup>

13 FIVs can be further classified into flutter,<sup>6</sup> vortex-induced vibration (VIV),<sup>7</sup> and  
 14 galloping,<sup>8</sup> each exhibiting distinct characteristics suitable for different environments.  
 15 These features enable the design of tailored TENGs to operate under low wind speeds,  
 16 adapt to a wide range of wind speeds, and accommodate compact configurations.<sup>9</sup>  
 17 Galloping is a self-excited vibration phenomenon driven by aerodynamic forces that  
 18 introduce negative damping. It typically occurs in structures with complex, irregular,  
 19 non-streamlined cross-sections, such as square, rectangular, and right-angled profiles.  
 20 Galloping enables continuous energy harvesting from ambient airflow, making it well-  
 21 suited for triboelectric nanogenerators (TENGs).<sup>10,11</sup> As an early demonstration of FIV-  
 22 driven TENGs, the flag-shaped design stands out as a representative prototype.<sup>12,13</sup>  
 23 Building on this concept, Yu *et al.*<sup>14</sup> designed an elastically assisted TENG with self-  
 24 charge excitation, in which the flutter film was encapsulated with aluminum foil to  
 25 mitigate environmental interference. Son *et al.*<sup>15</sup> proposed a dual-flagpole TENG with  
 26 grooved structures to enable bidirectional wind energy harvesting. Yue *et al.*<sup>16</sup>  
 27 developed a “mortise-and-tenon” micro-structured flutter film, achieving a power  
 28 density of 455.932 mW/cm<sup>2</sup> and an output voltage of 7.5 kV at a wind speed of 7.9 m/s

1 using an array of flag-shaped TENGs. Furthermore, the output performance of flag-  
 2 shaped TENGs can be enhanced by placing a bluff body upstream to reinforce the  
 3 downstream flow field.<sup>17,18</sup> Dong *et al.*<sup>19</sup> placed bluff bodies with different metasurfaces  
 4 upstream of the flag-shaped TENG, resulting in enhanced power output and improved  
 5 linearity in wind speed sensing. However, all these devices require the flutter films to  
 6 be directly exposed to the atmosphere, leading to two critical durability challenges: (1)  
 7 output degradation due to prolonged exposure to environmental stressors (humidity,  
 8 temperature fluctuations, and airborne particulates), and (2) gradual mechanical failure  
 9 (edge curling, wrinkling, and ultimately tearing) resulting from repetitive deformation  
 10 and friction-induced polarization.

11 To address the above-mentioned challenges, numerous studies have explored the  
 12 use of bluff body vibrations to drive TENGs in recent years.<sup>20,21</sup> Those designs are  
 13 typically realized by either attaching a bluff body to the free end of a cantilever beam  
 14 or springs, both forming elastically suspended oscillator systems. When stable fluid  
 15 flows over the bluff body, fluid-structure interaction occurs.<sup>22</sup> Such systems allow for  
 16 the integration of TENGs inside bluff bodies.<sup>23</sup> A widely adopted strategy utilizes  
 17 rolling beads to build the TENG, where the rolling beads can effectively convert the  
 18 bluff body's vibrational energy into electrical power. As an alternative approach, Guo  
 19 *et al.*<sup>24</sup> installed a switch-shuttling TENG inside a cylindrical-hexagram bluff body and  
 20 provided a self-powered water quality monitoring solution. However, this approach  
 21 increases the bluff body's mass, which in turn alters its dynamics.<sup>25</sup> In addition, the  
 22 inherent geometric constraints of bluff bodies may limit the design flexibility of the  
 23 TENGs. This limitation can be circumvented by using non-invasive space integration  
 24 strategies, with one simple and effective approach being the rigid attachment of TENGs  
 25 to the exterior of the bluff body or the cantilever beam. For instance, Kim *et al.*<sup>26</sup>  
 26 mounted a free-standing-sliding (FS) mode TENG and a contact-separation (CS) mode  
 27 TENG on the top of a cylindrical bluff body. Zhang *et al.*<sup>27</sup> installed a CS-mode TENG  
 28 near a cantilever beam, utilizing beam vibration to excite the device. However, this

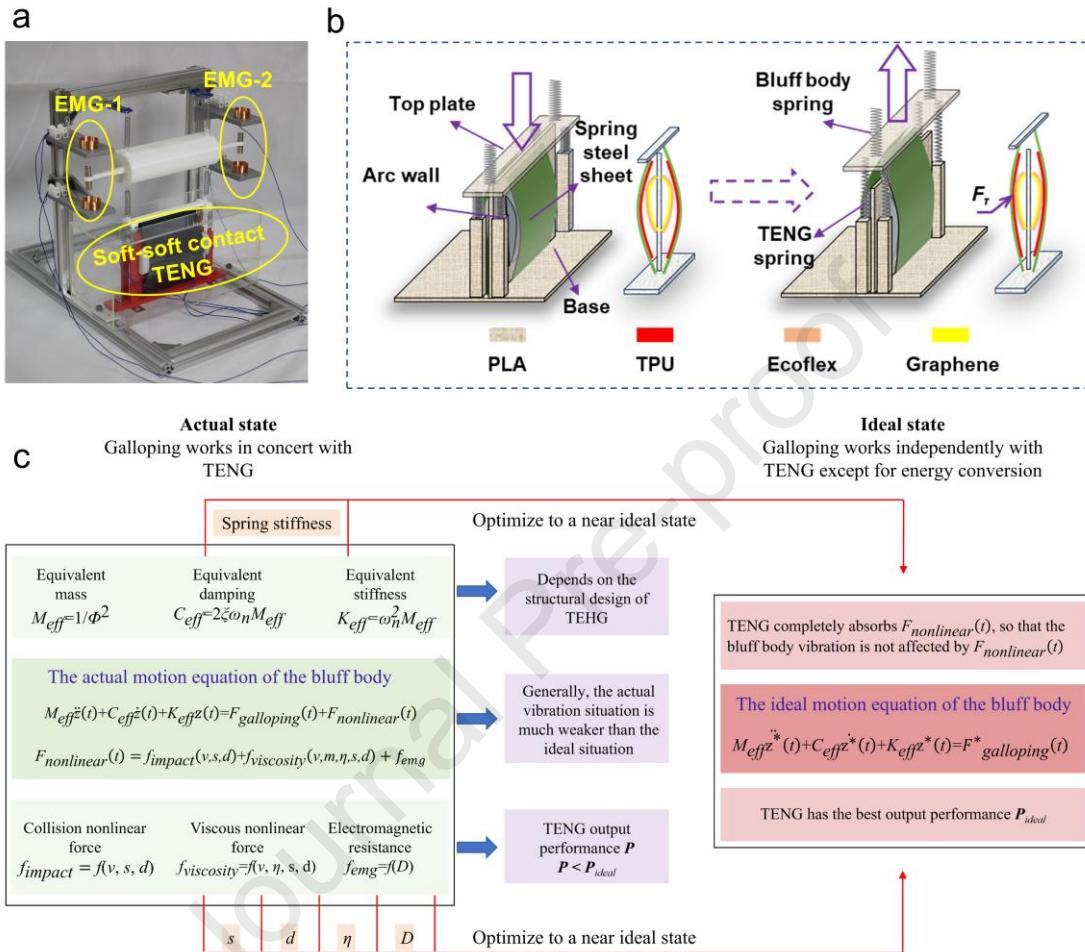
1 setup inevitably leads to collision-induced damage, degrading the output performance  
 2 of the TENG. An alternative and effective approach is to employ spring connections  
 3 between the bluff body and the TENG.<sup>28,29</sup> Such elastically coupled systems demand  
 4 synchronization between TENG's operation and the bluff body's vibration while being  
 5 affected by intricate nonlinear factors.<sup>30-32</sup>

6 In this study, we present a high-performance triboelectric-electromagnetic hybrid  
 7 generator driven by galloping (NG-TEHG), capable of harvesting wind energy across  
 8 a tunable range. This study investigates the effects of the system stiffness, configuration  
 9 geometry, and triboelectric material viscosity on the performance of the NG-TEHG. An  
 10 interesting finding is that by changing the viscosity of the triboelectric material, the  
 11 NG-TEHG achieves an adjustable wind energy harvesting range. The operational range  
 12 of the NG-TEHG first increases and then locks into a safe region (2.29-7.80 m/s)  
 13 following the bluff body's vibration. The peak and average power outputs of this NG-  
 14 TEHG reach approximately 10 mW and 3.34 mW, indicating 2.14-fold and 18.55-fold  
 15 improvements, respectively, compared to purely triboelectric and electromagnetic  
 16 designs.<sup>28</sup> Due to its excellent performance, the NG-TEHG can power a wireless  
 17 temperature sensor stably.

## 18 **2. Structural Configuration and Optimization Strategy**

19 Galloping is a common form of flow-induced vibration, typically characterized by  
 20 periodic oscillations when the wind speed exceeds a critical threshold. The oscillation  
 21 amplitude increases with increasing wind speed. This facilitates efficient wind energy  
 22 harvesting across a wide wind speed range. Building on this unique property and aiming  
 23 to enhance structural scalability, we propose an elastically attached bluff body-TENG  
 24 configuration, as shown in **Fig. 1(a)**. This configuration consists of a windward TENG  
 25 (W-TENG) and a leeward TENG (L-TENG), with the details illustrated in **Fig. S1** in  
 26 the Supplementary Information showing that the TENG unit in the NG-TEHG operates  
 27 within an independent compartment. This design mitigates spatial constraints imposed  
 28 by the bluff body's volume and prevents direct rigid impacts on it.<sup>33</sup> In addition, there

1 are two electromagnetic generators (EMGs) positioned on both sides of the bluff body  
 2 to further enhance the output performance. The parameters of the EMGs are listed in  
 3 **Tables S1 and S2** in the Supplementary Information.



5 **Fig.1 Structure design and optimization strategies of the NG-TEHG.** (a) Photograph  
 6 of the prototyped NG-TEHG, (b) Schematic diagram of the NG-TEHG, (c) Optimization  
 7 strategy for the NG-TEHG.

8 The output performance of FIV-TENGs can be generally enhanced by optimizing  
 9 the geometry of the bluff body, elastic suspension parameters, and triboelectric material  
 10 properties.<sup>34-36</sup> Beyond these factors, for TENGs designed to operate within  
 11 compartments elastically connected to the bluff body, the intricate coupling effects,  
 12 particularly those arising from segmented springs and nonlinear damping during TENG  
 13 operation, remain underexplored.<sup>37,38</sup> In this study, we systematically investigate these  
 14 effects in the NG-TEHG, with detailed schematics of the TENG unit shown in **Fig. 1(b)**.  
 15 A 50  $\mu\text{m}$  thick TPU film is coated on the spring steel sheet, serving as the positive

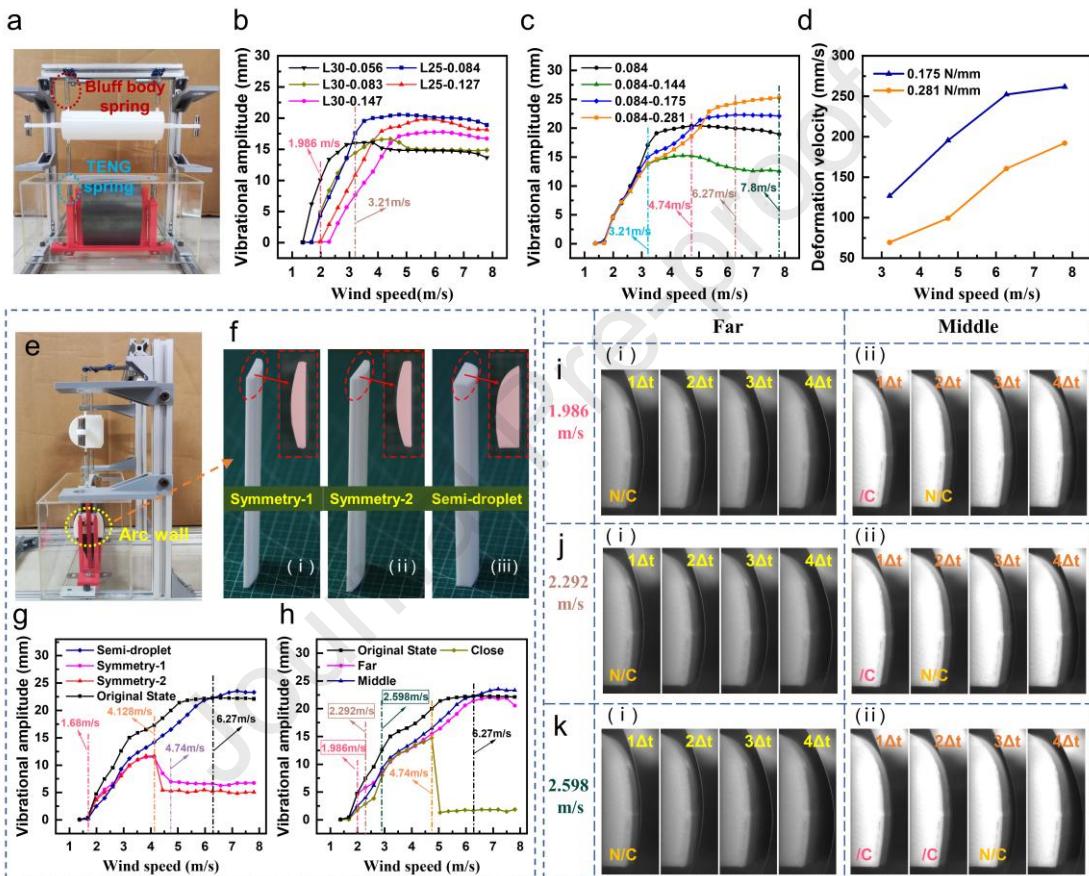
1 triboelectric material, and a high-performance flexible Ecoflex foam film is laminated  
2 onto the arc wall. The TENG unit functions as a spring-like element and is connected  
3 in series with the bluff body via real springs. This configuration enables the spring steel  
4 sheets to strike the arc wall in response to the bluff body's vibration, thereby activating  
5 TENG operation. As shown in **Fig. S2** in the Supplementary Information, contact  
6 between the spring steel sheet and arc wall induces charge transfer via the triboelectric  
7 effect, generating positive and negative charges on the TPU and Ecoflex foam films,  
8 respectively. Subsequently, during the separation process, electrons flow from the  
9 graphene electrode to the spring steel sheet, generating a current in the external circuit.  
10 When the separation distance reaches its maximum, the potential difference between  
11 the two electrodes also reaches its peak. As the spring steel sheet tends to reestablish  
12 contact with the arc wall, a reverse current is generated, and the potential difference  
13 between the two electrodes gradually reduces to zero. Once full contact is reestablished,  
14 the cycle is completed. This working cycle repeats continuously during the vibration of  
15 the bluff body, thus generating an alternating current that matches the vibration  
16 frequency. Through analysis of both bluff body geometry optimization and TENG  
17 material structure, it is concluded that during the operation of the NG-TEHG, the  
18 motion of the bluff body is controlled and influenced by three sets of nonlinear coupling  
19 forces: the aerodynamic force, the force from TENG unit, and the forces from two EMG  
20 units.

21 The nonlinear forces may alter the FIV mode, thereby affecting the efficiency of  
22 galloping. These nonlinear forces primarily include the impact force from spring steel  
23 sheets and viscous forces arising from the contact-separation process of the triboelectric  
24 material. These nonlinear effects can be changed by controlling four key parameters:  
25 the surface shape ( $s$ ) of the arc wall, the initial distance ( $d$ ) between the friction surfaces,  
26 the viscosity ( $\eta$ ) of triboelectric material, and the initial gap ( $D$ ) between the magnet  
27 and the coil. By systematically adjusting these parameters, the output performance of  
28 the NG-TEHG can be progressively optimized, and the specific optimization strategy

1 is shown in **Fig. 1(c)**. Ideally, the TENG unit is expected to remain in resonance with  
 2 the bluff body's galloping motion, thereby facilitating stable and efficient conversion  
 3 of mechanical energy into electricity.

### 4 3. Results and Discussion

#### 5 3.1. Structural tuning and dynamic enhancement of the NG-TEHG



6 **Fig.2 Structural optimization process of the NG-TEHG.** (a) Front-view photograph of  
 7 the NG-TEHG. Red dashed box: bluff body spring; blue dashed box: TENG spring. (b)  
 8 (c) Vibration amplitude responses of the bluff body suspended using different springs. Legend:  
 9 L length - spring stiffness. (d) Vibration amplitude responses in response to variations in  
 10 the TENG springs. Legend: bluff body spring stiffness-TENG spring stiffness. (e) Side-view  
 11 photograph of the NG-TEHG. Yellow dashed box: arc wall. (f) Three arc wall configurations  
 12 with different surface shapes. (g) Vibration amplitude responses of the three arc wall  
 13 configurations. (h) Vibration responses under varying initial distances between the spring  
 14 steel sheet and arc wall. (i-k) High-speed snapshots of spring steel sheet-wall interactions  
 15 at wind speeds of 1.98 m/s, 2.29 m/s, and 2.59 m/s, showing collision (/C) and no collision  
 16 (N/C) states at different initial distances.

1 The dynamics of any practical FIV systems are governed by their corresponding  
 2 mechanical structures. In the proposed NG-TEHG, the equivalent mass and damping  
 3 are primarily determined by the bluff body and the TENG unit, while the stiffness is  
 4 governed by the springs. As shown in **Fig. 2(a)**, the equivalent stiffness of the galloping  
 5 system can be adjusted by changing the bluff body spring and TENG spring. Five types  
 6 of bluff body springs and three types of TENG springs were tested to optimize galloping  
 7 dynamics (see Supplementary **Table S3** for spring parameters). To evaluate the impact  
 8 of different bluff body springs on galloping behavior, the bluff body was tested without  
 9 connection to the TENG, and the obtained time-history responses are presented in **Fig.**  
 10 **2(b)**. The results show that the NG-TEHG achieves optimal bluff body vibration when  
 11 springs with a length of 25 mm and a stiffness of 0.084 N/mm are used, exhibiting both  
 12 a lower onset wind speed and the ability to sustain large-amplitude vibrations at high  
 13 wind speeds. It is worth noting that when the TENG was connected, the arc wall was  
 14 not installed, thereby eliminating collision-induced nonlinear forces and viscous  
 15 nonlinear forces. As shown in **Fig. 2(c)**, among the three TENG springs tested, the  
 16 spring with a stiffness of 0.144 N/mm performed the worst. The TENG spring, with a  
 17 stiffness of 0.175 N/mm, produced weaker bluff body vibrations than the 0.281 N/mm  
 18 spring at wind speeds below 5.05 m/s, but exhibited stronger vibrations at wind speeds  
 19 above this threshold. For a more detailed comparison of the two TENG springs, we  
 20 monitored the deformation of the spring steel sheets at four wind speeds: 3.21 m/s, 4.74  
 21 m/s, 6.27 m/s and 7.80 m/s (deformation images are provided in Supplementary **Fig.**  
 22 **S3**). Using the operating frequency of the TENG at corresponding wind speeds, we  
 23 calculated the deformation velocities of the spring steel sheets of the TENG springs, as  
 24 shown in **Fig. 2(d)**. These results indicate that the TENG spring with a stiffness of 0.175  
 25 N/mm more effectively converts the bluff body's vibration energy.

26 The surface shape ( $s$ ) of the arc wall influences the nonlinear collision behavior  
 27 when the spring steel sheet impacts it. As shown in **Fig. 2(e, f)**, three configurations  
 28 were designed for optimization: two symmetric profiles (Supplementary **Fig. S4**) and

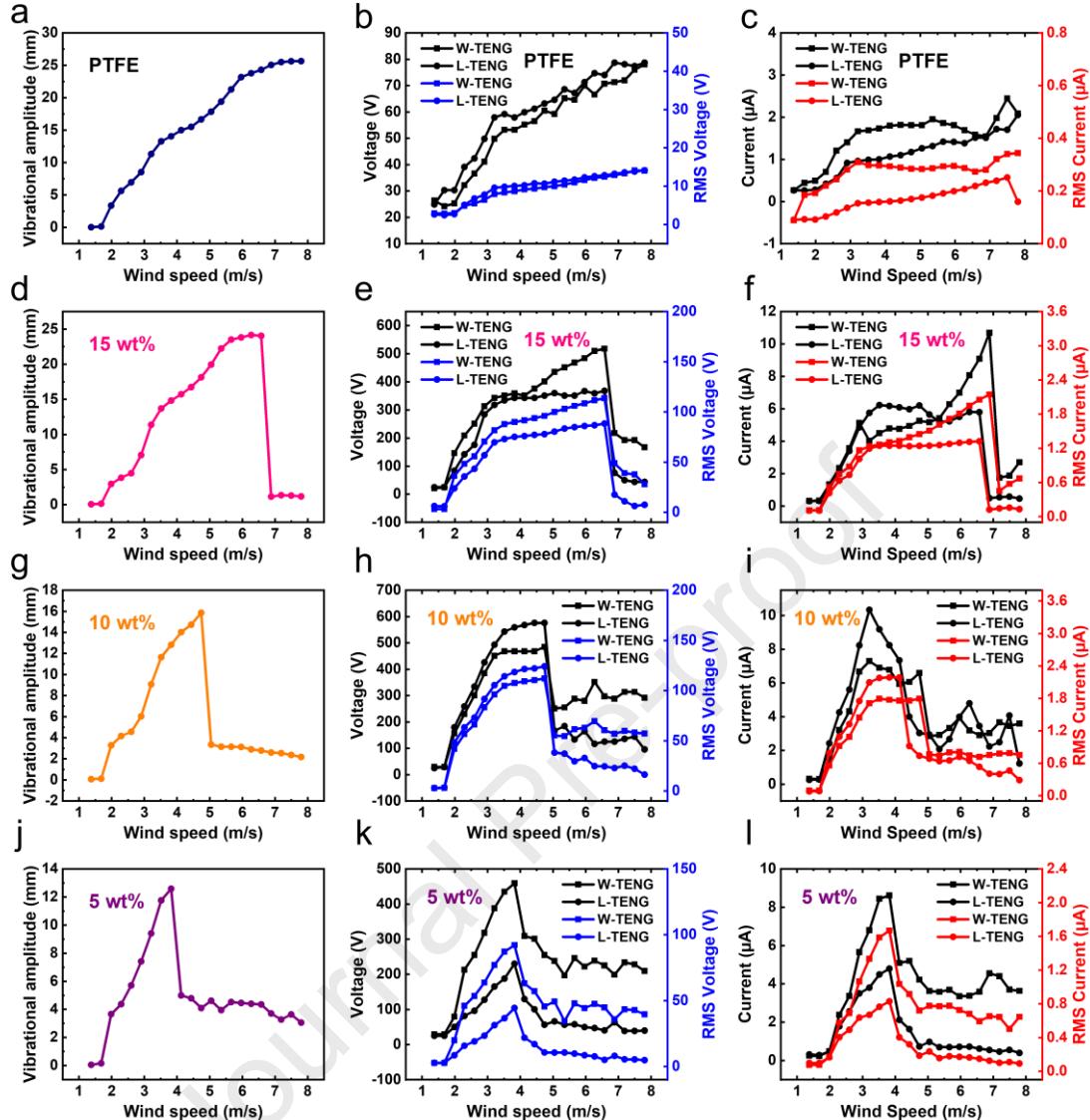
1 one asymmetric semi-droplet shape. Symmetric surfaces are generally considered more  
 2 compatible with the deformation of spring steel sheets, as they increase the contact area  
 3 and thereby enhance nonlinear collision forces. However, experimental results show  
 4 that both symmetric surfaces lead to a sharp decline in galloping vibration beyond  
 5 4.13 m/s, followed by a plateau, which suppresses the vibration under large wind speeds,  
 6 as shown in **Fig. 2(g)**. In contrast, the semi-droplet surface shape can sustain galloping,  
 7 with amplitudes beyond 6.27 m/s surpassing those observed in the initial non-collision  
 8 state. Therefore, this shape was adopted in the design of the arc wall.

9 In addition to the surface profile of the arc wall, it is also essential to optimize the  
 10 initial distance ( $d$ ) between the spring steel sheet and the arc wall. Three initial distances,  
 11 namely, “Close”, “Middle”, and “Far”, were defined by adjusting the thickness of the  
 12 arc wall. As shown in **Fig. 2(h)**, when the initial distance is set to the closest position  
 13 (Close), the galloping amplitude drops sharply at 4.74 m/s, significantly limiting the  
 14 operational wind range of the NG-TEHG. When the initial distances are set to Middle  
 15 and Far, the system can maintain galloping. At wind speeds above 2.59 m/s, the  
 16 galloping amplitude at the Middle distance consistently exceeds that at the Far distance.  
 17 Conversely, below 2.59 m/s, the galloping amplitude at the Middle distance remains  
 18 lower than at the Far distance. **Fig. 2(i, j, k)** illustrates the contact timing sequence  
 19 between the spring steel sheet and the arc wall at two initial distances under wind speeds  
 20 of 1.98 m/s, 2.29 m/s, and 2.59 m/s. Earlier contact signifies a lower onset wind speed,  
 21 which is favorable for wind energy harvesting. At wind speeds below 2.59 m/s, no  
 22 collision (N/C) occurs at the Far distance, whereas at the Middle distance, collisions  
 23 (/C) occur as early as 1.98 m/s. This suggests that a Middle initial distance between the  
 24 spring steel sheet and the semi-droplet-shaped arc wall not only promotes TENG  
 25 operation at lower wind speeds but also results in stronger galloping responses.

### 26 **3.2. Material viscosity-controlled bandwidth tuning in the NG-TEHG**

27 The optimized NG-TEHG ensures that the bluff body maintains galloping across

1 a wind speed range from 1.68 m/s to 7.80 m/s despite the presence of nonlinear collision  
2 effects between the spring steel sheet and the arc wall. Similar to the nonlinear collision  
3 effects, when a thermoplastic polyurethanes (TPU) film is attached to the spring steel  
4 sheet, and an Ecoflex foam film is attached to the arc wall surface, the TENG introduces  
5 nonlinear viscous effects due to material viscosity  $\eta$  during operation, thereby enabling  
6 the NG-TEHG to achieve a tunable operational wind speed range. To further explore  
7 the impact of material viscosity on the NG-TEHG, four sets of materials with different  
8 viscous properties were fabricated: transparent TPU and PTFE film, frosted TPU film  
9 and three types of Ecoflex foam films with varying adhesion levels. The foaming  
10 microsphere doping concentrations in these Ecoflex films were 15 wt%, 10 wt%, and 5  
11 wt%, respectively. By adjusting the doping concentration of the foaming microspheres,  
12 triboelectric materials with different adhesion characteristics were prepared.  
13 Triboelectric materials with different viscosities were prepared by varying the doping  
14 concentration of foamed microspheres. As the foaming degree increases, the viscosity  
15 decreases; notably, the frosted TPU film exhibits nearly no viscosity compared to the  
16 transparent TPU film. For comparative analysis, transparent TPU and PTFE films were  
17 also tested. When the PTFE film comes into contact with the transparent TPU film, the  
18 interaction produces minimal viscous force, making it an ideal reference for comparison.



1 **Fig.3 Influence of material viscosity on the NG-TEHG performance.** (a, b, c) Vibration  
2 amplitude of the bluff body and output voltage and current of the TENG when PTFE is used  
3 as the triboelectric material. (d, e, f) Vibration amplitude of the bluff body and output voltage  
4 and current of the TENG when the 15 wt% Ecoflex foam film is used. (g, h, i) Vibration  
5 amplitude of the bluff body and output voltage and current of the TENG when the 10 wt%  
6 Ecoflex foam film is used. (j, k, l) Vibration amplitude of the bluff body and output voltage  
7 and current of the TENG when the 5 wt% Ecoflex foam film is used.

8 As the viscosity of the triboelectric material increases, the cutoff wind speed  
9 exhibits a decreasing trend, as shown in **Fig. 3(a, d, g, j)**. The cutoff wind speed was  
10 observed at 6.58 m/s, 4.74 m/s, and 3.82 m/s, respectively. This trend is attributed to  
11 the increased difficulty in achieving separation after contact between the spring steel  
12 sheet and arc wall, leading to a mismatch between TENG operation and bluff body

1 vibration. When the galloping amplitude exceeds a certain threshold, the TENG  
 2 undergoes maximum deformation, and the viscous force between the triboelectric  
 3 materials reaches its peak. This directly disrupts the motion of the spring steel sheet,  
 4 preventing the TENG from maintaining resonance with the bluff body's galloping  
 5 motion. At this stage, the galloping vibration mode becomes entirely disrupted, and the  
 6 fluid-structure interaction with wind is significantly weakened or lost, leading to  
 7 reduced or decreased vibration. As shown in **Fig. 3(b, c)**, the TENG exhibits  
 8 significantly lower output performance when PTFE is used as the triboelectric material  
 9 compared to when Ecoflex foam film is used. This demonstrates that the working range  
 10 of the NG-TEHG can be effectively tuned by adjusting the material viscosity. When  
 11 using 15 wt% Ecoflex foam film, the TENG achieves an optimal operating range of  
 12 1.98–6.58 m/s, with the maximum open-circuit voltage reaching 518 V for the W-  
 13 TENG and 368 V for the L-TENG, as shown in **Fig. 3(e, f)**. When using 10 wt% Ecoflex  
 14 foam film, the W-TENG achieves a maximum open-circuit voltage of 576 V, an RMS  
 15 voltage of 124.1 V, a maximum short-circuit current of 10.33  $\mu$ A, and an RMS current  
 16 of 2.18  $\mu$ A. The output of TENGs is typically short and pulsed, with voltages mostly  
 17 far below the peak level, making it ineffective for evaluating the energy harvesting  
 18 performance. Therefore, a comprehensive and scientific performance evaluation must  
 19 include both metrics. The L-TENG achieves a maximum open-circuit voltage of 485 V,  
 20 an RMS voltage of 112.1 V, a maximum short-circuit current of 7.3  $\mu$ A, and an RMS  
 21 current of 1.79  $\mu$ A. Beyond the optimal operating range of 1.98–4.74 m/s, the bluff body  
 22 continues to maintain small-amplitude vibrations, allowing the TENG to remain  
 23 functional. This endows the NG-TEHG with the dual advantages: 1) The NG-TEHG  
 24 exhibits a broad operational bandwidth, functioning effectively at wind speeds ranging  
 25 from 1.98 m/s to 4.74 m/s. 2) At wind speeds exceeding 4.74 m/s, the amplitude of the  
 26 bluff body no longer increases indefinitely due to the nonlinear resistance limiting its  
 27 motion. Instead, it demonstrates small-amplitude oscillations, thereby protecting the  
 28 structure, as demonstrated in **Fig. 3(h, i)**. When using 5 wt% Ecoflex foam film, the

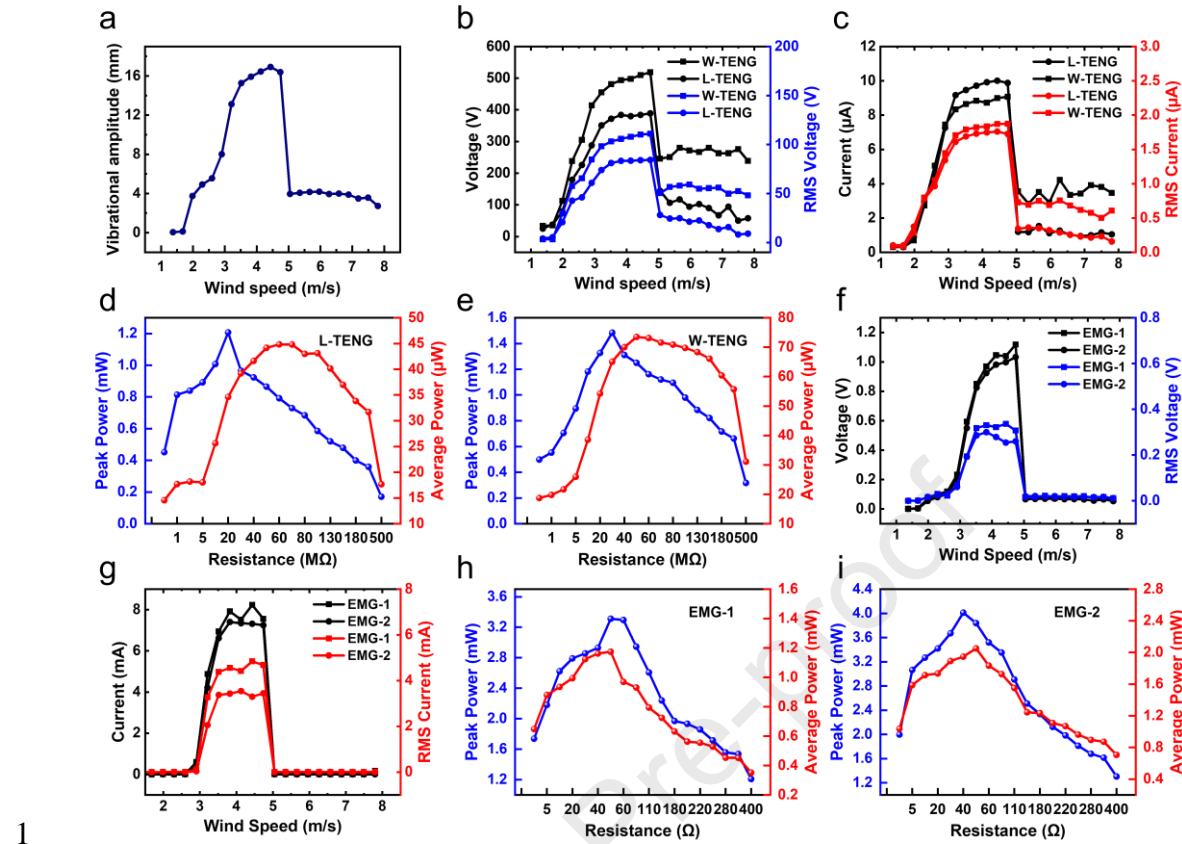
1 TENG's optimal working range is limited to 1.98-3.822 m/s, with only small-amplitude  
 2 vibrations (below 6 mm) maintained between 3.82 m/s and 7.80 m/s. Consequently,  
 3 the output performance is notably lower than that achieved using 10 wt% Ecoflex foam  
 4 film, as shown in **Fig. 3(k, l)**. In the W-TENG, the spring steel sheet and the arc wall  
 5 are subjected to an additional force induced by wind pressure, which enhances the  
 6 contact force between the two triboelectric materials. As a result, the open-circuit  
 7 voltage of the W-TENG is higher than that of the L-TENG. Based on the experimental  
 8 results, it can be concluded that as the content of foaming microspheres gradually  
 9 decreases (from 15 wt% to 5 wt%), the amplitude of the bluff body gradually decreases,  
 10 and the wind speed corresponding to the maximum amplitude also reduces. Meanwhile,  
 11 the maximum output voltage exhibits a trend of increasing first and then decreasing.

### 12 **3.3. Output characteristics and practical applications of the NG-TEHG**

13 Through a quantitative analysis of the three factors, i.e., the surface shape ( $s$ ) of  
 14 the arc wall, the initial distance ( $d$ ) between the friction surfaces, and the viscosity ( $\eta$ )  
 15 of the triboelectric material, the fundamental operating principle of the NG-TEHG was  
 16 established. As shown in **Fig. 3(g, h, i)**, when using 10 wt% Ecoflex foam film, the bluff  
 17 body achieves a maximum galloping amplitude of 15.8 mm, and the TENG generates  
 18 considerable outputs at wind speeds below 4.74 m/s. When the wind speed exceeds 4.74  
 19 m/s, the TENG continues to operate with a certain level of efficiency, sustained by the  
 20 small-amplitude vibrations of the bluff body, thereby ensuring a wide working range.  
 21 Therefore, the 10 wt% Ecoflex foam film and frosted TPU film were selected as the  
 22 triboelectric materials for the NG-TEHG.

23 Additionally, to prevent collisions between the magnet and coil and to ensure that  
 24 the bar magnet can move freely within the coil at high amplitudes, the distance  $D$   
 25 between the top of the bar magnet and the coil was set to 20 mm. As shown in **Fig. 4(a)**,  
 26 the bluff body vibration of the NG-TEHG first increases in amplitude with rising wind  
 27 speed, then decreases, and eventually stabilizes at a small amplitude of approximately

1 4 mm once the wind speed exceeds 4.74 m/s. Under this condition, the open-circuit  
2 voltage and short-circuit current outputs from the TENG unit are shown in **Fig. 4(b, c)**.  
3 It demonstrates a steadily increasing output performance in the wind speed range of  
4 1.98–4.74 m/s. Specifically, the W-TENG reaches a maximum open-circuit voltage of  
5 519 V (RMS voltage of 111 V), a maximum short-circuit current of 9.07  $\mu$ A (RMS  
6 current of 1.87  $\mu$ A). In comparison, the L-TENG achieves a maximum open-circuit  
7 voltage of 389 V (RMS voltage of 84.3 V), a maximum short-circuit current of 9.88  $\mu$ A  
8 (RMS current of 1.73  $\mu$ A). However, in the wind speed range of 4.74–7.80 m/s, a  
9 sudden reduction in bluff body amplitude leads to a sharp decline in the TENG output.  
10 The open-circuit voltage of the W-TENG drops to about 270 V, while that of the L-  
11 TENG falls to approximately 100 V. Due to the displacement of the bluff body in the  
12 direction of the incoming wind, the contact distance in the W-TENG decreases while  
13 that in the L-TENG increases. As a result, the W-TENG delivers a consistently higher  
14 output voltage than the L-TENG. After approximately 100,000 operating cycles, the L-  
15 TENG retains 95% of its initial output performance, while the W-TENG retains 73%,  
16 as shown in Supplementary **Fig. S5**.



1 **Fig.4 Output performance of the NG-TEHG.** (a) Vibration amplitude of the bluff body  
2 versus the wind speed. (b, c) Open-circuit voltage and short-circuit current of the TENG  
3 versus the wind speed. (d, e) Power outputs of the L-TENG and W-TENG versus the load  
4 resistance. (f, g) Open-circuit voltage and short-circuit current of the EMG under varying  
5 wind speeds. (h, i) Power outputs of EMG-1 and EMG-2 versus the load resistance.

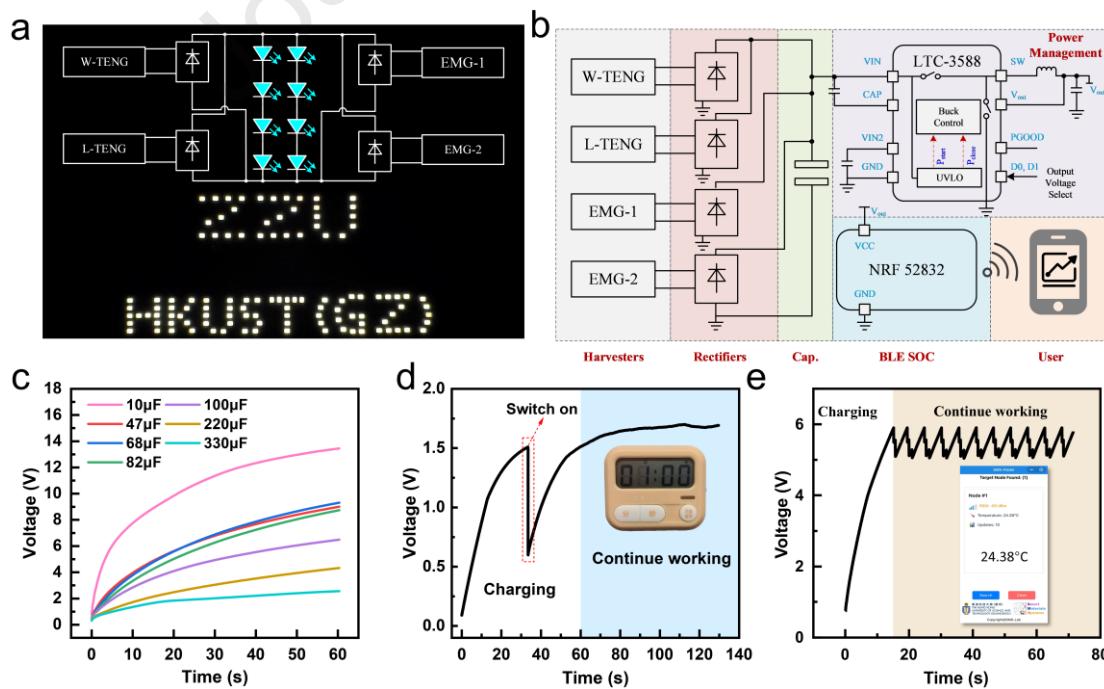
7 The power output of the TENG was evaluated at a wind speed of 4.74 m/s, and the  
8 results are offered in Supplementary **Fig. S6**. The output power of TENG was calculated  
9 as shown in **Fig. 4(d, e)**. The W-TENG achieved a peak power of 1.48 mW at a load  
10 resistance of 30 MΩ, with a max RMS power of 73.48 μW at 50 MΩ. In comparison,  
11 the L-TENG attained a peak power of 1.2 mW at a load resistance of 20 MΩ, and the  
12 corresponding maximum RMS power was 44.83 μW at 60 MΩ. The reason for the  
13 above results is that the triboelectric material employed in this study is flexible Ecofoam  
14 film, which is easily deformable and covered with microsphere structures on its surface.  
15 When left and right TENGs were exposed to different pressures, the deformation of the  
16 microsphere structures varied considerably, therefore causing alterations in the internal  
17 resistance accordingly. In addition, the open-circuit voltage and short-circuit current of

1 the EMG as a function of wind speed are presented in **Fig. 4(f, g)**. The EMG started to  
 2 operate at a wind speed of 2.29 m/s, with a small amplitude of approximately 5 mm,  
 3 during which the power output remained limited. In the wind speed range of 2.90–4.74  
 4 m/s, the galloping amplitude exceeded 8 mm, placing the voltage output in an optimal  
 5 region. Under these conditions, EMG-1 achieved a peak-to-peak voltage of 1.11 V, an  
 6 RMS voltage of 0.33 V, a peak current of 8.23 mA, and an RMS current of 4.84 mA.  
 7 EMG-2 generated a peak-to-peak voltage of 1.03 V, an RMS voltage of 0.3 V, a peak  
 8 current of 7.4 mA, and an RMS current of 3.54 mA. As the wind speed continued to  
 9 rise, the sudden reduction in galloping amplitude caused the EMG to cease operation.  
 10 The maximum power output of the EMG at a wind speed of 4.74 m/s is shown in **Fig.**  
 11 **4(h, i)**. EMG-1 delivered a peak power of 3.31 mW and a maximum RMS power of  
 12 1.17 mW at a load resistance of  $50\Omega$ . EMG-2 produced a peak power of 4.01 mW at a  
 13 load resistance of  $40\Omega$  and a maximum RMS power of 2.05 mW at  $50\Omega$ . The observed  
 14 results can be attributed to two primary factors: First, nonlinear perturbations between  
 15 wind resistance and the springs led to inconsistencies between EMG-1 and EMG-2.  
 16 Secondly, the electromagnetic induction voltage signals, which are highly sensitive to  
 17 both magnet position and velocity, resulted in irregular measurements. Therefore, the  
 18 optimal impedance of EMG-1 and EMG-2 differs during peak power measurement, but  
 19 aligns during RMS power measurement. Overall, at a wind speed of 4.74 m/s, the NG-  
 20 TEHG achieved a maximum total power output of 10 mW and an RMS power output  
 21 of 3.34 mW.

22 In the application, the four energy harvesting units of the NG-TEHG are rectified  
 23 to realize AC-to-DC conversion and then connected in parallel to power external loads,  
 24 as illustrated in **Fig. 5(a)**. Benefiting from the high-voltage output of the TENG units,  
 25 the NG-TEHG is capable of driving high-voltage devices. As shown in **Fig. 5(a)**, it can  
 26 light up two display boards, i.e., one spelling “ZZU” with 32 LED bulbs and the other  
 27 spelling “HKUST(GZ)” with 81 LED bulbs. Each LED operates at a forward voltage  
 28 of about 1.5 V, demonstrating the system’s ability to power multi-LED configurations

effectively. **Fig. 5(b)** presents the circuit schematic of a custom-designed, self-powered IoT temperature sensing node, solely powered by the NG-TEHG. The harvested outputs are first rectified and stored in a capacitor. An energy management unit (LTC3588) then regulates the voltage to provide stable power to a BLE-enabled IoT node (NRF52832), enabling real-time temperature sensing and wireless data transmission.

The capacitor charging rate serves as a key indicator of the output performance. As shown in **Fig. 5(c)**, at the wind speed of 4.74 m/s, the NG-TEHG was able to charge capacitors of 10  $\mu$ F, 47  $\mu$ F, 68  $\mu$ F, 82  $\mu$ F, 100  $\mu$ F, 220  $\mu$ F, and 330  $\mu$ F to voltages of 13.45 V, 3.17 V, 9 V, 9.31 V, 8.74 V, 6.49 V, 4.32 V, and 2.56 V, respectively, within 60 seconds. In addition, **Fig. 5(d, e)** demonstrates the NG-TEHG's capability to power microelectronic devices at a wind speed of 4.74 m/s. Using only the TENG units, an electronic timer can be continuously powered after being charged for 60 seconds. Additionally, the NG-TEHG successfully powered the BLE-enabled IoT node (**Fig. 5(b)**), which represents a more advanced application, continuously after only 15 seconds of charging, as shown in **Fig. 5(e)**. The excellent performance of the NG-TEHG underscores its potential as a sustainable power solution for remote monitoring and paves the way for the development of intelligent self-powered wireless sensing systems.



**Fig.5 Self-powered applications of the NG-TEHG.** (a) Demonstration of the NG-TEHG

1 simultaneously powering two LED display boards: "ZZU" with 32 bulbs and "HKUST(GZ)"  
 2 with 81 bulbs. (b) Schematic of the self-powered IoT sensing node. Harvested energy is  
 3 rectified, stored in a capacitor, and regulated by LTC3588 to power the BLE-enabled IoT  
 4 node for wireless temperature sensing. (c) Testing the NG-TEHG's capability to charge  
 5 different capacitors. (d) Using the NG-TEHG to power an electronic timer. (e) Using the  
 6 NG-TEHG to power a wireless temperature sensor after just 15 seconds of charging.

## 7 **4. Conclusion**

8 This work reports a novel nonlinear galloping-driven triboelectric-electromagnetic  
 9 hybrid generator (NG-TEHG), designed to efficiently harvest low-speed wind energy.  
 10 With a dual-mass architecture comprising bluff body-TENG elastic attachments, the  
 11 system achieves tunable wind speed ranges and high electrical output through nonlinear  
 12 parameter optimization. Through a comprehensive analysis of the nonlinear coupling  
 13 effects between the TENG units and the galloping structure, we uncover critical insights  
 14 for optimizing its energy harvesting performance. Experimental results show that the  
 15 NG-TEHG could operate efficiently over a broad wind speed range of 2.29-7.80 m/s,  
 16 achieving a peak power output of 10 mW and an RMS power of 3.34 mW at 4.74 m/s.  
 17 At a wind speed of 4.74 m/s, the open-circuit voltage of the W-TENG unit reaches a  
 18 maximum of 519 V, and that of the L-TENG unit reaches 389 V, delivering a peak power  
 19 of 10 mW and an average power of 3.34 mW. These values represent improvements of  
 20 2.14 times and 18.55 times, respectively, compared to a conventional TENG device.  
 21 This level of power output supports continuous operation of a wireless temperature  
 22 sensor, demonstrating the system's potential as a reliable and sustainable power source  
 23 for self-powered devices. In general, the NG-TEHG presents a significant step forward  
 24 in the development of self-powered systems, paving the way for more efficient and  
 25 versatile energy harvesters in low-speed wind environments.

## 26 **Experimental section**

### 27 **Design sources of three different surface-shaped arc walls**

28 As shown in Supplementary **Fig. S2**, when the surface of the arc wall fits perfectly

1 with the spring steel sheet's deformation, the contact area of the TENG is maximized,  
 2 leading to the strongest nonlinear effects on the bluff body of the NG-TEHG. However,  
 3 the impact of these effects on the bluff body vibration remains uncertain. To investigate  
 4 the influences of different surface shapes, dynamic deformation images of the spring  
 5 steel sheet, captured by a Megapixel Network Surface Array Camera, were analyzed  
 6 frame by frame. Distance markers were applied along both the X and Y axes on the  
 7 images, and the distance data were fitted to determine the cross-sectional shape of the  
 8 deformed spring steel sheet. This analysis led to the design of two symmetric profiles,  
 9 Symmetry-1 and Symmetry-2, that correspond to the vibration of the spring steel sheet.  
 10 For comparison, an asymmetric curved surface, resembling the shape of a water droplet  
 11 (semi-droplet-shaped surface), was also designed. This surface shape induces a lateral  
 12 force  $F_\tau$  during contact, which can leverage the material's viscosity characteristics to  
 13 enhance the output performance, though it may result in increased material wear.

#### 14 **Preparation process of the Ecoflex foam film**

15 The preparation of the Ecoflex foam film begins by weighing equal amounts of  
 16 Ecoflex<sup>TM</sup>00-30 A and Ecoflex<sup>TM</sup>00-30 B solutions (Smooth-on, America).  
 17 Subsequently, the material is doped by adding thermally expandable microspheres  
 18 (120DU25, Polychem, America) with different mass fractions into the Ecoflex mixed  
 19 solution. The mixture is stirred thoroughly to ensure uniform dispersion of the  
 20 microspheres throughout the solution. During the curing process, it is crucial to ensure  
 21 adequate degassing for optimal foaming. To remove any small bubbles formed during  
 22 stirring, the Ecoflex-microsphere mixture undergoes vacuum treatment. The degassed  
 23 solution is then evenly applied to a pre-treated Polyvinyl chloride (PVC) sheet,  
 24 followed by pressing with a glass plate to ensure a uniform film thickness. After curing  
 25 at room temperature for 5 hours, the film is placed in a vacuum oven at 100°C for 30  
 26 minutes to complete the whole process, resulting in an Ecoflex foam film with a  
 27 protruding microsphere surface structure.

#### 28 **Fabrication of the NG-TEHG**

1        The bluff body, the top plate of the TENG, and the base were fabricated using 3D  
 2 printing with PLA material. The bluff body springs, TENG springs, spring steel sheets,  
 3 TPU films, PTFE films, magnets, and coils were all commercially sourced. The TENG  
 4 and the bluff body were mounted on an aluminum profile frame, and the TENG was  
 5 fully enclosed in an acrylic shell.

6        **Resources availability**

7        **Materials availability**

8        \*\*\*

9        **Data and code availability**

10        \*\*\*

11        **Supporting Information**

12        **Author contributions**

13        \*\*\*

14        **Acknowledgments**

15        \*\*\*

16        **Conflict of interest**

17        The authors declare that they have no known competing financial interests or personal  
 18 relationships that could have appeared to influence the work reported in this paper.

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Highlights:

1. A nonlinear galloping-driven triboelectric–electromagnetic hybrid generator (NG-TEHG) is developed for efficient low-speed wind energy harvesting.
2. Nonlinear structural and material tuning enables wideband operation (2.29–7.80 m/s) with a peak output of 10 mW.
3. The NG-TEHG powers LEDs and a wireless IoT sensor, demonstrating practical self-powered applications.

**Declaration of interests**

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

The authors declare the following financial interests/personal relationships which may be considered as potential competing interests: